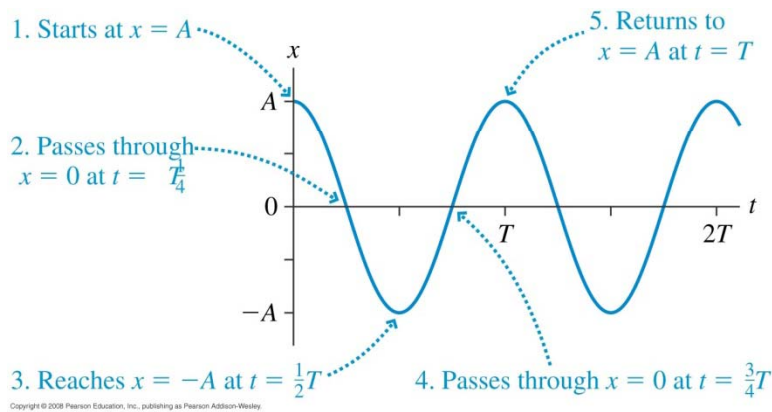


P2211K

11 / 16 / 2010

# Oscillations, vibrations, and simple harmonic motion (SHM)

**Definition of SHM:** Sinusoidal (or cosine) oscillations



**Functional form:**  $x(t) = A\cos(\omega t + \phi)$ , where

$x(t)$  = position

$\omega$  = angular frequency (radians / s) =  $2\pi f$

$f$  = frequency in Hertz (Hz)

$A$  = amplitude

$\phi$  = phase (to come later)

**Other relations:**

$f$  = frequency = repetitions / time

$T$  = Period = time / repetition

$T = 1/f$

- Note that the motion has gone through one complete cycle when  $\omega t = 2\pi$ . Thus the functional form can be rewritten in many forms:

$$x(t) = A\cos(\omega t + \phi) = A\cos(2\pi f t + \phi) = A\cos\left(\frac{2\pi t}{T} + \phi\right)$$

- Also, note that velocity and acceleration are related to position by

$$x(t) = A\cos(\omega t + \phi)$$

$$v(t) = \frac{dx}{dt} = -\omega A\sin(\omega t + \phi)$$

$$a(t) = \frac{dv}{dt} = \frac{d^2x}{dt^2} = -\omega^2 A\cos(\omega t + \phi)$$

## So how do we get SHM?

- **Basic answer:** spring-like forces, *i.e.*, ones described by a linear restoring behavior---  $F = -kx$ .
- **Why is this important?**
  1. SHM gives the most basic version of a **periodic** motion;
  2. The characteristic of a linear restoring force is a good approximation of more complex forces (one example is the simple pendulum);
  3. Chemical bonds, and thus their vibration behaviors, can be approximated by this type of force;
  4. The deflection (or bending) behavior of structural units (beams, etc.) can be described to a good approximation by these forces;
  5. etc.
- How do we know spring-like forces produce SHM? Look at the “ $F = ma$ ” relation:

$$ma(t) = -kx(t) \Rightarrow m \frac{d^2 x(t)}{dt} = -kx(t) \Rightarrow \frac{d^2 x(t)}{dt} + \left( \frac{k}{m} \right) x(t) = 0$$

*For this type of differential equation, the general solution is*

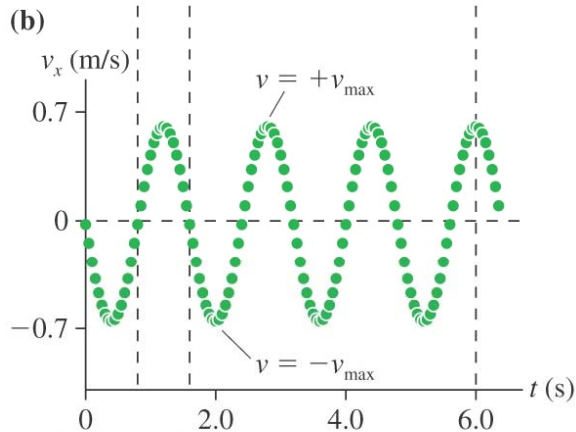
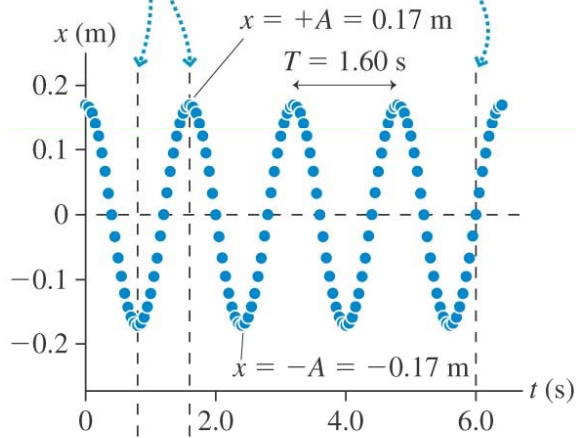
$$x(t) = A \sin(\omega t) + B \cos(\omega t) \quad \text{with } \omega^2 = \frac{k}{m}$$

*(A and B are amplitude parameters determined by the initial conditions.)*

- Thus, this demonstrates that SHM is the result of the linear restoring force.

# Dynamics of SHM

(a) The speed is zero when  $x = \pm A$ . The speed is maximum as the object passes through  $x = 0$ .



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$$x(t) = A \cos(\omega t + \phi)$$

$$v(t) = \frac{dx}{dt} = -\omega A \sin(\omega t + \phi)$$

$$a(t) = \frac{dv}{dt} = \frac{d^2x}{dt^2} = -\omega^2 A \cos(\omega t + \phi)$$

- From the relations for  $x(t)$ ,  $v(t)$ , and  $a(t)$ , we see that the **maximum displacement is  $A$** , the **maximum speed is  $\omega A$** , and the **maximum acceleration is  $\omega^2 A$** . Furthermore, from the relations between sin and cos functions, **the maximum displacement occurs when the speed is zero**, and the **maximum speed occurs when the displacement is zero**.
- Also, we know from previous discussions that the spring-mass system is **conservative** so that

$$\Delta K + \Delta U = 0 \Rightarrow E_T = \frac{1}{2}mv^2 + \frac{1}{2}kx^2$$

and that

$$E_T = \frac{1}{2}m v_{\max}^2 = \frac{1}{2}kA^2$$

or

$$E_T = \frac{1}{2}m v_{\max}^2 = \frac{1}{2}m \omega^2 A^2$$

**Problem 14-13.** A 200 g mass attached to a horizontal spring oscillates at a frequency of 2.0 Hz. At  $t = 0$ , the mass is at  $x = 5.0$  cm and has  $v_x = -30.0$  cm/s.

Determine:

- a. The period.  $T = 1/f = 0.5$  sec
- b. The angular frequency.  $\omega = 2\pi f = 12.57$  rad/s
- c. The amplitude.  $E_T = \frac{1}{2}kA^2$ ; Need  $k = m\omega^2 = 31.6$  N/m &  
 $E_T = \frac{1}{2}kx^2 + \frac{1}{2}mv^2 = 0.0485$  J, so  
 $A = (x^2 + mv^2/k)^{1/2} = (x^2 + v^2/\omega^2)^{1/2} = 5.54$  cm
- d. The phase constant.  $x(t) = A\cos(\omega t + \phi)$ , so  $x(0) = A\cos(\phi)$  &  
 $\phi = \cos^{-1}[x(0)/A] = \cos^{-1}[5/5.54] = 25.5^\circ = 0.445$  rad
- e. The maximum speed.  $E_T = \frac{1}{2}m(v_{\max})^2$ , so  $v_{\max} = (2E_T/m)^{1/2} = 69.6$  cm/s  
also,  $v_{\max} = A\omega$
- f. The maximum acceleration.  $a_{\max} = \omega^2 A = 875.3$  cm/s<sup>2</sup>
- g. The total energy.  $E_T = 0.0485$  J
- h. The position at  $t = 0.40$  s.  $x(0.40) = 5.54\cos(12.57 * 0.40 + 0.445)$  cm  
 $= 3.23$  cm

## Energy in Simple Harmonic motion (SHM):

- Review Example 14.5, p. 419

## Dynamics of SHM:

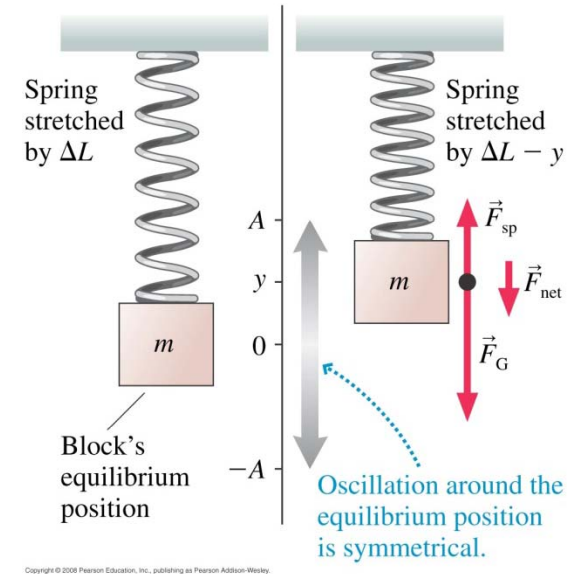
- Review Example 14.6, pp. 422-423

## Vertical Oscillations: How does $g$ affect things?

- No effect on frequency
- Shifts  $x = 0$  position by  $x = mg / k$

## The simple pendulum: how does it execute SHM?

- For SHM,  $ma = -kx = md^2x/dt^2$
- This has the form:  $d^2\mathbf{g}/dt^2 = -(k/m)\mathbf{g}$ , where  $\mathbf{g}$  represents a coordinate [such as linear position ( $x$ ), arc length ( $s$ ), angular position ( $\theta$ ), etc.] and leads to  $\omega^2 = k/m$ .
- For the simple pendulum (as discussed in section 14.5, pp. 425-426), the force law, with  $s = L\theta$  and the small-angle approximation that  $\sin\theta \sim \theta$ , becomes  $mL(d^2\theta/dt^2) = \sim -mg\theta$ , which has the form  $d^2\theta/dt^2 = \sim - (g/L)\theta$ .
- Thus, by comparison with the spring-mass system,  $\omega^2 = g/L$ .
- Alternately, we can note that the potential energy for the exact SHM system has the form  $U = \frac{1}{2}kx^2$ , which is a parabola. For the pendulum,  $U_g = mgL(1 - \cos \theta)$ , which is a circle. A circle  $\sim$  a parabola for small  $\theta$ .



### Damped harmonic motion:

- Basic idea: when set into “motion,” the oscillations will continue until the energy is removed (or dissipated).
- Typical damping is proportional to speed (or the 1<sup>st</sup> derivative of the generalized coordinate **g** mentioned before).
- “Damping” removes the vibrational energy.

<http://www.lon-capa.org/~mmp/applist/damped/d.htm>

### Driven oscillations:

- Basic idea: all objects have a “natural frequency” and a damping factor
- Energy can be put into the system @ the natural frequency.
- If the damping is low enough that the energy put in is more than that taken out by damping, the vibrational amplitude will build up, possibly to catastrophic levels.

<http://www.walter-fendt.de/ph14e/resonance.htm>

Driven harmonic motion—voice breaking glass:

<http://www.youtube.com/watch?v=eTWDEsGIPO8&NR=1&feature=fvwp> 1:30

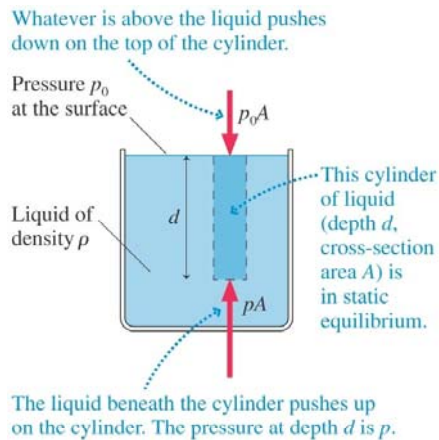
<http://www.youtube.com/watch?v=lZD8ffPwXR0&feature=related> 1:42

Driven harmonic motion—Tacoma Narrows bridge:

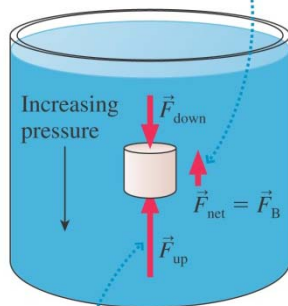
<http://www.youtube.com/watch?v=j-zczJXSxw>

## Ch. 15: Fluids & elasticity

- **Fluids:** fluids are not rigid and “can take on the shape of their container.” Gases & liquids are obvious examples. An important characteristic of many liquids is that they are (nearly) incompressible—examples are water and hydraulic fluids.



The net force of the fluid on the cylinder is the buoyant force  $\vec{F}_B$ .

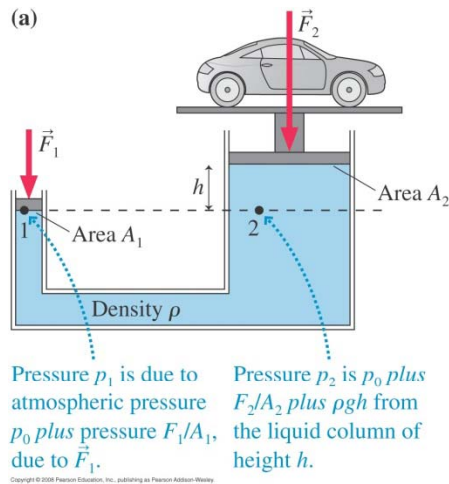


$F_{\text{up}} > F_{\text{down}}$  because the pressure is greater at the bottom. Hence the fluid exerts a net upward force.

- **Pressure:**  $P = \text{force} / \text{area} = F/A$ ; Units =  $\text{N/m}^2$  ( $1 \text{ N/m}^2 = 1 \text{ Pa}$ );  $\text{lb/in}^2$ ;  $\text{mm Hg}$  ( $1 \text{ mm Hg} = 1 \text{ Torr}$ );  $\text{cm H}_2\text{O}$ ; etc
- **(mass) Density:**  $\rho = \text{mass} / \text{volume} = M / V$ ; units =  $\text{kg/m}^3$ ;  $\text{g/cm}^3$ ,  $\text{kg/liter}$ ;  $\text{g/ml}$ , etc. ( $1 \text{ ml} = 1 \text{ cm}^3 = 1 \text{ cc}$ )
- **Pressure at a depth in a fluid**—is due to the weight of the fluid above it: Above the space of area  $A$  is a column of fluid with height  $d$  (the depth); the weight of this column is  $Mg = \rho(Ad)g$ , which leads to  $P = F/A = Mg/A = \rho g d$
- **Archimedes's principle, buoyancy, and floating**—arise from the pressure-at-a-depth result: for any 3-dimensional object, the pressure at its bottom, which is at a greater depth, is larger than the pressure at its top.
- **The expression for the buoyant force:** At the bottom, the pressure is  $\rho g d_{\text{bottom}}$ , at the top it is  $\rho g d_{\text{top}}$ , and their difference is  $\Delta P = \rho g(d_{\text{bottom}} - d_{\text{top}})$ . ( $d_{\text{bottom}} - d_{\text{top}}$ ) = height of the object. Multiplying both sides by  $A$  gives the net force  $F_{\text{net}} = \Delta P(A) = \rho g A \text{ height}$ . Because  $A \cdot \text{height}$  is the volume of the object, this becomes  $F_{\text{buoyant}} = \rho g V_{\text{obj}} = \rho g V_{\text{disp}}$ , where  $v_{\text{disp}}$  is the volume of fluid displaced.



- **Pressure in fluids & Pascal's Principle:** “A change in pressure at one point in an incompressible fluid appears undiminished at all points in the fluid.”
- **Hydraulics:** come from application of Pascal's principle



- In the situation sketched to the side, applying force  $F_1$  to the piston of area  $A_1$  creates pressure  $P = F_1/A_1$ . This is “transmitted” to the other piston of area  $A_2$  to create force  $F_2 = PA_2 = (F_1/A_1)A_2 = F_1(A_2/A_1)$ . Obviously, if  $(A_2/A_1) > 1$ , then  $F_2 > F_1$ . ( $F_2/F_1 = A_2/A_1$  is the “mechanical advantage” of the system.)
- However, to accomplish this, the smaller piston must “pump” a volume of fluid into the larger cylinder  $V = d_2 A_2$ . To do so, the smaller piston must move  $d_1 = V/A_1 = d_2(A_2/A_1)$ , which is **greater than  $d_2$**  by the same ratio that  **$F_2$  is greater than  $F_1$** .
- If the two pistons are not at the same level ( $h_2 \neq h_1$ ), and this naturally occurs when they move, then there is a pressure difference on both sides due to the “pressure-at-a-depth” result:  $\Delta P = \rho_{fluid}(h_2 - h_1)g$ . Holding an object with the pistons at different heights requires adding this pressure to the lower piston.

- **Work through example 15.7 on page 455.**

## **Assignment:**

- **Review Chapter 13 with a focus on the more general form for gravitational potential energy and the “escape velocity,” section 13-5.**
- **Read Chapters 14 & 15**
- **Begin Reading Chapter 16**